

LABORATORY ASSIGNMENT

After reviewing Chapter of the lecture notes write a MATLAB program that calculates axial displacements and forces in a bar.

The bar is fixed on the left side, subjected to a continuous load q and a concentrated force P at the right end. Use linear shape functions and the CALFEM instructions *assem* and *solveq*.

Determine the elements of the stiffness matrix and load vector using symbolic operations with *diff* and *int*.

Calculate natural frequencies and mode shapes using the *eigen* procedure.

In the report, submit to the instructor:

- the program code, and
- a brief report containing:
 - plots of displacements, axial force,
 - the first four mode shapes and corresponding frequencies,
 - and a convergence plot of the displacement at the end of the bar as a function of the number of degrees of freedom.

For a selected finite element, use the solution vector from the program to extract the corresponding degrees of freedom and write down the formula for the approximate solution in that element **by hand**.

Make sure the input data and results are realistic.

Algorithm:

1. After declaring **syms** x , define the input data: q (as a function of x), E , A , L , P .
2. Define the number of elements **Nel** and compute:
 - the number of degrees of freedom **Ndof**, and
 - the vector of node coordinates **coord**.

3. Initialize the global stiffness matrix **KG**, mass matrix **MG**, and the global load vector **fG** as zero.
4. Insert the value of the concentrated force **P** into the appropriate entry of the vector **fG**.
5. Loop over elements (**iel = 1:Nel**)

- a. Select node coordinates **x1** and **x2**
- b. Define linear shape functions **f1** and **f2**, and compute their derivatives (symbolically)
- c. Form matrices **B** and **N**
- d. Compute the element stiffness matrix **Kel** and load vector **Pel** (using symbolic integration)
- e. Use the **assem** function to assemble:

[KG, fG] = assem([iel, iel, iel+1], KG, Kel, fG, Pel);

- f. Compute the element mass matrix **Mel** (symbolically)
- g. Use **assem** again to update the mass matrix:

MG = assem([iel, iel, iel+1], MG, Mel);

6. Create a boundary condition vector **bc** of size 1×2: the number of the known DOF (displacement) and its value.
7. Solve the system of equations with the kinematic conditions using **solveq**:

u = solveq(KG, fG, bc);

8. Use the **plot** command to visualize the displacement approximation.
9. Loop over elements (**i = 1:Nel**):

- a. Select node coordinates **x1** and **x2**
- b. **Extract** DOF values from vector **u**
- c. Compute derivatives of the shape functions
- d. Write the formula for axial force **S** in the element
- e. Use **fplot** to visualize the force approximation:

fplot(S, [x1, x2]), hold on

10. Use the **eigen** procedure to compute frequencies and mode shapes:

[La, Egv] = eigen(KG, MG, bc(:, 1));

11. Plot the first four mode shapes. Example for the first one:

Freq = sqrt(La)/(2*pi);

figure(3)

plot(coord, Egv(:,1)), grid on

MATLAB Code Skeleton (to copy and complete):

% AXIAL LOADING OF A BAR

clear;

syms x

%% INPUT DATA

E =

A =

rho =

L =

q =

P =

%% DISCRETIZATION

Nel =

Ndof = Nel + 1;

coord = linspace(0, L, Ndof);

```
%% INITIALIZATION
```

```
KG = zeros(Ndof, Ndof);
```

```
MG =
```

```
fG =
```

```
fG(end) = % Apply concentrated force
```

```
%% ELEMENT LOOP
```

```
for iel = 1:Nel
```

```
    x1 = coord(iel);
```

```
    x2 =
```

```
    phi1 = (x - x2) / (x1 - x2);
```

```
    phi2 =
```

```
    N = [phi1, phi2];
```

```
    B =
```

```
    Kel =
```

```
    Pel =
```

```
    [KG, fG] = assem([iel, iel, iel+1], KG, Kel, fG, Pel);
```

```
    Mel =
```

```
    MG =
```

```
end
```

```
%% SOLVE SYSTEM
```

```
bc =
```

```
u = solveq(KG, fG, bc);
```

```
figure(1)

plot

xlabel('x'); ylabel('Displacement'); grid on
```

```
%% AXIAL FORCE
```

```
figure(2), hold on

for iel = 1:Nel

    x1 = coord(iel);

    x2 =

    u1 = u(iel);

    u2 =

    dphi1 = 1 / (x1 - x2);

    dphi2 =

    S =

    fplot(S, [x1, x2]), hold on

end

xlabel('x'); ylabel('Axial Force'); grid on
```

```
%% NATURAL FREQUENCIES & MODES
```

```
[La, Egv] = eigen(KG, MG, bc(:,1));

Freq = sqrt(La) / (2 * pi);
```

```
figure(3)

plot(coord, Egv(:,1)), grid on

xlabel('x'); ylabel('1st Mode Shape')
```